

# 3. Variációszámítás (anal3)

Kiegészítő anyag, szorgalmi házi feladatok, Matlab

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## 1 Geodetikus görbék

### 1.1 Geodetikus görbék általános felületen

Adott  $S \subset \mathbb{R}^3$  felület

$$S = \left\{ s(u, v) = (x(u, v), y(u, v), z(u, v)) \mid (u, v) \in D \right\} \tag{1}$$

ahol  $D \subset \mathbb{R}^2$ , illetve adottak  $P_1 = (u_1, v_1), P_2 = (u_2, v_2) \in S$  pontok. A célunk, hogy kiszámoljuk azt a

$$\Gamma = \{ \gamma(t) \mid t \in [t_1, t_2] \} \subset S \tag{2}$$

minimális hosszúságú görbét, melyre  $\gamma(t_1) = P_1$  és  $\gamma(t_2) = P_2$ .

Mivel  $\Gamma \subset S$ , legyen a görbe paraméterezése

$$\gamma(t) \underset{\substack{u=u(t) \\ v=v(t)}}{=} s(u(t), v(t)) = \begin{pmatrix} x(u(t), v(t)) \\ y(u(t), v(t)) \\ z(u(t), v(t)) \end{pmatrix}. \tag{3}$$

Tudjuk, hogy a  $\Gamma$  görbe hossza

$$S(\Gamma) = \int_{\Gamma} dl = \int_{t_1}^{t_2} \|\dot{\gamma}(t)\| dt = \int_{t_1}^{t_2} \sqrt{\dot{x}^2 + \dot{y}^2 + \dot{z}^2} dt \tag{4}$$

ahol a láncszabály miatt

$$\begin{aligned} \dot{x} &= \frac{d}{dt} x(u(t), v(t)) = \frac{\partial x}{\partial u} \dot{u} + \frac{\partial x}{\partial v} \dot{v} \\ \dot{y} &= \frac{d}{dt} y(u(t), v(t)) = \frac{\partial y}{\partial u} \dot{u} + \frac{\partial y}{\partial v} \dot{v} \\ \dot{z} &= \frac{d}{dt} z(u(t), v(t)) = \frac{\partial z}{\partial u} \dot{u} + \frac{\partial z}{\partial v} \dot{v}. \end{aligned} \tag{5}$$

Tehát

$$\begin{aligned} \dot{x}^2 + \dot{y}^2 + \dot{z}^2 &\stackrel{(5)}{=} \left( \frac{\partial x}{\partial u} \dot{u} + \frac{\partial x}{\partial v} \dot{v} \right)^2 + \left( \frac{\partial y}{\partial u} \dot{u} + \frac{\partial y}{\partial v} \dot{v} \right)^2 + \left( \frac{\partial z}{\partial u} \dot{u} + \frac{\partial z}{\partial v} \dot{v} \right)^2 = \\ &= \underbrace{\left( \left( \frac{\partial x}{\partial u} \right)^2 + \left( \frac{\partial y}{\partial u} \right)^2 + \left( \frac{\partial z}{\partial u} \right)^2 \right)}_P \dot{u} + 2 \underbrace{\left( \frac{\partial x}{\partial u} \frac{\partial x}{\partial v} + \frac{\partial y}{\partial u} \frac{\partial y}{\partial v} + \frac{\partial z}{\partial u} \frac{\partial z}{\partial v} \right)}_Q \dot{u} \dot{v} + \\ &+ \underbrace{\left( \left( \frac{\partial x}{\partial v} \right)^2 + \left( \frac{\partial y}{\partial v} \right)^2 + \left( \frac{\partial z}{\partial v} \right)^2 \right)}_R \dot{v} = \dot{u}^2 P + \dot{u} \dot{v} Q + \dot{v}^2 R. \end{aligned} \tag{6}$$

Ekkor  $v$ -t felírhatjuk  $u$  függvényében, olyan  $v(u)$  függvényt keresve, melyre  $v(u_1) = v_1$  és  $v(u_2) = v_2$ .

$$\dot{u}^2 P + \dot{u} \dot{v} Q + \dot{v}^2 R \underset{\substack{v=v(u) \\ \dot{v}=v'\dot{u}}}{=} \dot{u}^2 P + \dot{u}^2 v' Q + \dot{u}^2 v'^2 R = \dot{u}^2 (P + v' Q + v'^2 R). \quad (7)$$

(4)-be visszahelyettesítve

$$\begin{aligned} S(\Gamma) &\stackrel{(7)}{=} \int_{t_1}^{t_2} \sqrt{\dot{u}^2 (P + v' Q + v'^2 R)} dt = \int_{t_1}^{t_2} \sqrt{P + v' Q + v'^2 R} \dot{u} dt \stackrel{du=\dot{u}dt}{=} \\ &= \int_{u_1}^{u_2} \sqrt{P + v' Q + v'^2 R} du. \end{aligned} \quad (8)$$

Tehát a minimalizálandó funkcionál

$$I(v) = \int_{u_1}^{u_2} \sqrt{P + v' Q + v'^2 R} du = \int_{u_1}^{u_2} F(u, v, v') du. \quad (9)$$

Hasonlóan felírhatjuk  $u$ -t  $v$  függvényében, ekkor a funkcionál

$$I(u) = \int_{v_1}^{v_2} \sqrt{u'^2 P + u' Q + R} dv = \int_{v_1}^{v_2} F(v, u, u') dv. \quad (10)$$

## 1.2 Geodetikus görbék egységgömbön

Legyen az egységgömb paraméterezése

$$s(\theta, \phi) = \begin{pmatrix} \sin \phi \cos \theta \\ \sin \phi \sin \theta \\ \cos \phi \end{pmatrix} \quad (11)$$

illetve

$$\gamma(t) = \begin{pmatrix} x(\theta(t), \phi(t)) \\ y(\theta(t), \phi(t)) \\ z(\theta(t), \phi(t)) \end{pmatrix} = \begin{pmatrix} \sin \phi(t) \cos \theta(t) \\ \sin \phi(t) \sin \theta(t) \\ \cos \phi(t) \end{pmatrix} \quad (12)$$

amiből

$$\begin{aligned} P &= \cos^2 \phi(t) \cos^2 \theta(t) + \cos^2 \phi(t) \sin^2 \theta(t) + \sin^2 \phi(t) = 1 \\ Q &= -2 \sin \phi(t) \cos \phi(t) \sin \theta(t) \cos \theta(t) + 2 \sin \phi(t) \cos \phi(t) \sin \theta(t) \cos \theta(t) = 0 \\ R &= \sin^2 \phi(t) \sin^2 \theta(t) + \sin^2 \phi(t) \cos^2 \theta(t) = \sin^2 \phi(t). \end{aligned} \quad (13)$$

Tehát (9) alapján a minimalizálandó funkcionál

$$I(\theta) = \int_{\phi_1}^{\phi_2} \sqrt{1 + \theta'^2(\phi) \sin^2 \phi} d\phi = \int_{\phi_1}^{\phi_2} F(\phi, \theta, \theta') d\phi. \quad (14)$$

Felírva az Euler-Lagrange egyenletet

$$\begin{aligned} L[\theta] &= \frac{\partial F}{\partial \theta} - \frac{d}{d\phi} \frac{\partial F}{\partial \theta'} = \frac{\partial}{\partial \theta} \sqrt{1 + \theta'^2 \sin^2 \phi} - \frac{d}{d\phi} \frac{\partial}{\partial \theta'} \sqrt{1 + \theta'^2 \sin^2 \phi} = \\ &= -\frac{d}{d\phi} \frac{\theta' \sin^2 \phi}{\sqrt{1 + \theta'^2 \sin^2 \phi}} = 0 \implies \frac{\theta' \sin^2 \phi}{\sqrt{1 + \theta'^2 \sin^2 \phi}} = c_1. \end{aligned} \quad (15)$$

(15)-ből kifejezve  $\theta'$ -t

$$\theta' = \frac{c_1}{\sqrt{\sin^4 \phi - c_1^2 \sin^2 \phi}} \quad (16)$$

amiből

$$\begin{aligned} \theta &= c_1 \int \frac{d\phi}{\sqrt{\sin^4 \phi - c_1^2 \sin^2 \phi}} = c_1 \int \frac{d\phi}{\sin^2 \phi \sqrt{1 - \frac{c_1^2}{\sin^2 \phi}}} = \\ &= c_1 \int \frac{d\phi}{\sin^2 \phi \sqrt{1 - c_1^2 - c_1^2 \cot^2 \phi}} \stackrel{\substack{\tau = \cot \phi \\ d\tau = -\frac{1}{\sin^2 \phi} d\phi}}{=} -c_1 \int \frac{d\tau}{\sqrt{1 - c_1^2 - c_1^2 \tau^2}} = \\ &= -\int \frac{\frac{c_1}{\sqrt{1 - c_1^2}}}{\sqrt{1 - \left(\frac{c_1 \tau}{\sqrt{1 - c_1^2}}\right)^2}} d\tau = \arccos \frac{c_1 \tau}{\sqrt{1 - c_1^2}} + c_2 = \arccos \frac{c_1 \cot \phi}{\sqrt{1 - c_1^2}} + c_2. \end{aligned} \quad (17)$$

Tehát

$$\cos(\theta - c_2) = \cos \theta \cos c_2 + \sin \theta \sin c_2 = \frac{c_1 \cot \phi}{\sqrt{1 - c_1^2}} \quad (18)$$

amiből

$$\sin \phi \cos \theta \cos c_2 + \sin \phi \sin \theta \sin c_2 - \frac{c_1 \cos \phi}{\sqrt{1 - c_1^2}} = 0. \quad (19)$$

Ekkor (12) alapján

$$x \cos c_2 + y \sin c_2 - \frac{zc_1}{\sqrt{1 - c_1^2}} = 0. \quad (20)$$

Azt látjuk, hogy a két pontot összekötő minimális hosszúságú felületi görbe egy olyan síkon fekszik, amely átmegy a két ponton és a kör középpontján (hiszen az origó kielégíti (20)-t), tehát a két ponton átmenő főkör a megoldás. Ezen felül a kezdeti feltételek (tehát a két pont) figyelembe vételével (17)-ből vagy (20)-ból könnyen kiszámolhatjuk a  $c_1, c_2$  konstansok értékét.

### 1.3 Kiegészítés

#### Euler-Lagrange egyenlet speciális felületekre

A (9)-ben szereplő funkcionálra felírva az Euler-Lagrange egyenletet

$$\begin{aligned} L[v] &= \frac{\partial F}{\partial v} - \frac{d}{du} \frac{\partial F}{\partial v'} = \frac{\partial}{\partial v} \sqrt{P + v'Q + v'^2R} - \frac{d}{du} \frac{\partial}{\partial v'} \sqrt{P + v'Q + v'^2R} = \\ &= \frac{\frac{\partial P}{\partial v} + v' \frac{\partial Q}{\partial v} + v'^2 \frac{\partial R}{\partial v}}{2\sqrt{P + v'Q + v'^2R}} - \frac{d}{du} \frac{Q + v'R}{\sqrt{P + v'Q + v'^2R}} = 0. \end{aligned} \quad (21)$$

Tekintsük azon speciális eseteket, amikor  $P, Q, R$  csak  $u$ -nak függvényei. Ekkor (21)-ből

$$-\frac{d}{du} \frac{Q + v'R}{\sqrt{P + v'Q + v'^2R}} = 0 \implies \frac{Q + v'R}{\sqrt{P + v'Q + v'^2R}} = c_1. \quad (22)$$

Ebből rendezéssel az alábbi másodfokú egyenletet kapjuk

$$v'^2 R(R - c_1^2) + v'Q(R - c_1^2) + \frac{1}{4}Q^2 - Pc_1^2 = 0 \quad (23)$$

amiből

$$v' = \frac{Q(c_1^2 - R) \pm \sqrt{Q^2(R - c_1^2)^2 - R(R - c_1^2)(Q^2 - 4Pc_1^2)}}{2R(R - c_1^2)}. \quad (24)$$

Tekintsük azt a további speciális esetet, amikor  $Q = 0$ .

$$v' = \frac{\sqrt{4R(R - c_1^2)Pc_1^2}}{2R(R - c_1^2)} = c_1 \sqrt{\frac{P}{R(R - c_1^2)}} \quad (25)$$

így

$$v = c_1 \int \sqrt{\frac{P}{R(R - c_1^2)}} du. \quad (26)$$

Láthatjuk, hogy (26)-ból az egységgömb esetében (13)-t figyelembe véve azonnal megkapjuk (16)-t és (17)-t.

## 2 Síkinga mozgásegyenletének levezetése Lagrange módszerrel

Egy elhanyagolható tömegű  $L$  hosszú fonál egyik végét egy stabil ponthoz rögzítjük, a másik végére pedig egy  $m$  tömegű pontszerű testet helyezünk. Az így kapott ingát adott kezdeti szögéből indítva szabadon engedjük az  $(x, z)$  síkban. Adjuk meg az inga mozgási és helyzeti energiáját, majd a legkisebb hatás elvét követve vezessük le az inga mozgásegyenletét.

- Az inga mozgási (kinetikus) energiája:

$$T = \frac{mv^2}{2} = \frac{mL^2\dot{\theta}^2}{2} \quad (27)$$

- Az inga helyzeti (potenciális) energiája:

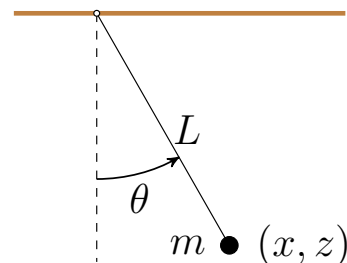
$$V = mgh = mgL(1 - \cos \theta) \quad (28)$$

Tehát a minimalizálandó funkcionál:

$$\int_{t_0}^{t_1} T - V dt, \quad \text{ahol } F(t, \theta, \dot{\theta}) = T - V = \frac{mL^2\dot{\theta}^2}{2} - mgL(1 - \cos \theta) \quad (29)$$

Az  $F(\cdot, \cdot, \cdot)$  függvényt a fizikában Lagrange függvénynek hívjuk (Lagrangian), és így jelöljük:  $\mathcal{L} = T - V$ . Ekkor az Euler-Lagrange egyenlet a következő lesz:

$$\mathcal{L}'_{\theta} - \frac{d}{dt} \mathcal{L}'_{\dot{\theta}} = -mgL \sin \theta - \frac{d}{dt} (mL^2\dot{\theta}) = -mgL \sin \theta - mL^2\ddot{\theta} = 0 \quad (30)$$



Ezért az inga mozgása a következő nemlineáris másodrendű differenciáegyenlettel írható le:

$$\ddot{\theta} + \omega^2 \sin \theta = 0, \quad \text{ahol } \omega = \sqrt{\frac{g}{L}}. \quad (31)$$

### 3 Crane model (rakodó darumodell)

Consider the following mathematical model of a crane machine. For the sake of simplicity, we restrict the motion of the carried weight to the  $(y, z)$  plane. In the model we identify the following time-dependent (state-) variables:

- $R = R(t)$  denotes the actual position of the car on the rail,
- $L = L(t)$  denotes the actual length of the wire,
- $\theta = \theta(t)$  denotes the actual angle of the wire with the vertical.

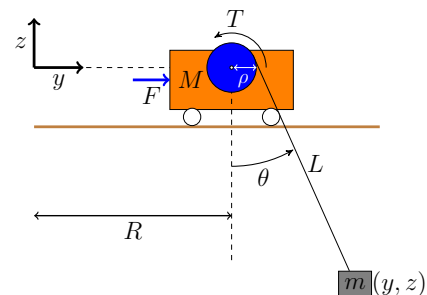


Figure 1. Kinetic model of the crane machine.

Known parameters (constants):

- $\rho$  is the radius of the pulley<sup>1</sup>. In the geometrical expression the radius of the pulley is neglected, eg. if the angle is  $\theta = 0$  and the position of the car is  $R = 0$  then the position of the weight along the  $y$  axis is considered to be  $R$ .
- $M$  is the mass of the car
- $m$  is the mass of the lifted weight
- $J$  is the moment of inertia<sup>2</sup> of the pulley.

Manipulated inputs are the following:

- $F$  is the driving force applied to the car
- $T$  is the torque applied to the pulley

Measured quantities:  $R(t)$  and  $L(t)$ .

Tekintse az ábrán látható rakodó darut. Az egyszerűség kedvéért a teher mozgását az  $(y, z)$  függőleges síkra korlátozzuk. Jelölje  $R$  a sínen mozgó kocsi  $y$  irányú pozícióját,  $L$  a sodrony hosszát és  $\theta$  a sodrony függőlegessel bezárt szögét. Ismert geometriai paraméter a sodronyt tekerceselő dob sugara, melyet  $\rho$ -val jelöltünk. (A geometriai összefüggésekben a dob sugarát elhanyagoljuk, azaz  $\theta = 0$  esetben  $R = y$ ). Szintén ismertek az inercia paraméterek:  $M$  a kocsi tömege;  $m$  a mozgatott teher tömege és  $J$  a sodronyt tekerceselő hajtás és a dob tehetetlenségi nyomatéka.

A beavatkozó jelek:

- $F$  a kocsira ható erő
- $T$  a sodronyt tekerceselő dobra ható forgatónyomaték

A mért kimeneti változók:  $R$  és  $L$ .

The system's kinetic energy is a composition of the followings:

1. the kinetic energy of  $M$  is  $T_M = \frac{M\dot{R}^2}{2}$
2. the kinetic energy of  $m$  is  $T_m = \frac{mv^2}{2}$
3. the kinetic energy of the pulley is  $T_J = \frac{J\dot{\theta}^2}{2} = \frac{J\dot{L}^2}{2\rho^2}$

The system's potential energy is the potential energy of  $m$ , that is  $V_m = mgL(1 - \cos \theta)$ .

The system's Lagrangian is

$$\mathcal{L} = T - V = \frac{M\dot{R}^2}{2} + \frac{mv^2}{2} + \frac{J\dot{L}^2}{2\rho^2} - mgL(1 - \cos \theta) \quad (32)$$

The velocity  $\mathbf{v}$  of  $m$  has the following components (see Figure 1):

$$\mathbf{v} = L\dot{\theta}\mathbf{e}_t + \dot{R}\mathbf{e}_x + \dot{L}\mathbf{e}_n \quad (33)$$

<sup>1</sup>sheave or drum: tekerceselő csiga vagy tekerceselő dob

<sup>2</sup>tehetetlenségi nyomaték

Since  $\mathbf{e}_t \perp \mathbf{e}_n$ , the square of the norm of  $\mathbf{v}$  can be computed in the following way

$$\begin{aligned} v &= \|\mathbf{v}\|^2 = \langle \mathbf{v}, \mathbf{v} \rangle = L^2 \dot{\theta}^2 + \dot{R}^2 + \dot{L}^2 + 2L\dot{\theta}\dot{R}\langle \mathbf{e}_t, \mathbf{e}_x \rangle + 2\dot{R}\dot{L}\langle \mathbf{e}_n, \mathbf{e}_x \rangle \\ &= L^2 \dot{\theta}^2 + \dot{R}^2 + \dot{L}^2 + 2L\dot{\theta}\dot{R} \cos \theta + 2\dot{R}\dot{L} \cos \left( \frac{\pi}{2} - \theta \right) \\ &= L^2 \dot{\theta}^2 + \dot{R}^2 + \dot{L}^2 + 2L\dot{\theta}\dot{R} \cos \theta + 2\dot{R}\dot{L} \sin \theta \end{aligned} \quad (34)$$

Therefore, the Lagrangian can be written in the following form:

$$\mathcal{L} = \frac{(M+m)\dot{R}^2}{2} + \frac{mL^2\dot{\theta}^2}{2} + \frac{m\dot{L}^2}{2} + mL\dot{\theta}\dot{R} \cos \theta + m\dot{R}\dot{L} \sin \theta + \frac{J\dot{L}^2}{2\rho^2} - mgL(1 - \cos \theta) \quad (35)$$

The Euler-Lagrange equations are the following:

$$\begin{aligned} 0 &= \frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{R}} - \frac{\partial \mathcal{L}}{\partial R} = \frac{d}{dt} \left( (M+m)\dot{R} + mL\dot{\theta} \cos \theta + m\dot{L} \sin \theta \right) \\ &= (M+m)\ddot{R} + m\dot{L}\dot{\theta} \cos \theta + mL\ddot{\theta} \cos \theta - mL\dot{\theta}^2 \sin \theta + m\ddot{L} \sin \theta + m\dot{L}\dot{\theta} \cos \theta \\ &= (M+m)\ddot{R} + 2m\dot{L}\dot{\theta} \cos \theta + mL\ddot{\theta} \cos \theta - mL\dot{\theta}^2 \sin \theta + m\ddot{L} \sin \theta \end{aligned} \quad (A1)$$

Second equation:

$$\begin{aligned} 0 &= \frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{L}} - \frac{\partial \mathcal{L}}{\partial L} = \frac{d}{dt} \left( m\dot{L} + m\dot{R} \sin \theta + \frac{J\dot{L}}{\rho^2} \right) - mL\dot{\theta}^2 - m\dot{\theta}\dot{R} \cos \theta + mg(1 - \cos \theta) \\ &= \left( m + \frac{J}{\rho^2} \right) \ddot{L} + m\ddot{R} \sin \theta + \underbrace{m\dot{R}\dot{\theta} \cos \theta}_{\text{red}} - mL\dot{\theta}^2 - \underbrace{m\dot{\theta}\dot{R} \cos \theta}_{\text{red}} + mg(1 - \cos \theta) \\ &= \left( m + \frac{J}{\rho^2} \right) \ddot{L} + m\ddot{R} \sin \theta - mL\dot{\theta}^2 + mg(1 - \cos \theta) \end{aligned} \quad (A2)$$

Third equation:

$$\begin{aligned} 0 &= \frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\theta}} - \frac{\partial \mathcal{L}}{\partial \theta} = \frac{d}{dt} \left( mL^2\dot{\theta} + mL\dot{R} \cos \theta \right) + mL\dot{\theta}\dot{R} \sin \theta - m\dot{R}\dot{L} \cos \theta + mgL \sin \theta \\ &= 2mL\dot{L}\dot{\theta} + mL^2\ddot{\theta} + \underbrace{m\dot{L}\dot{R} \cos \theta}_{\text{red}} + mL\ddot{R} \cos \theta - \underbrace{mL\dot{R}\dot{\theta} \sin \theta}_{\text{red}} + \underbrace{mL\dot{\theta}\dot{R} \sin \theta}_{\text{red}} - \underbrace{m\dot{R}\dot{L} \cos \theta}_{\text{red}} + mgL \sin \theta \\ &= 2mL\dot{L}\dot{\theta} + mL^2\ddot{\theta} + mL\ddot{R} \cos \theta + mgL \sin \theta \end{aligned}$$

Dividing by  $L$  we get:

$$0 = 2m\dot{L}\dot{\theta} + mL\ddot{\theta} + m\ddot{R} \cos \theta + mg \sin \theta \quad (A3)$$

In equilibrium point the torque is  $T_0 = mg\rho$ . Let  $T = T_0 + \tau$  be the net torque. Furthermore, we consider an  $F$  external force applying to  $M$ . Therefore, the equations of motion could be written as follows:

$$\begin{aligned} F &= (M+m)\ddot{R} + 2m\dot{L}\dot{\theta} \cos \theta + mL\ddot{\theta} \cos \theta - mL\dot{\theta}^2 \sin \theta + m\ddot{L} \sin \theta \\ -\frac{T}{\rho} &= -\frac{mg + \tau}{\rho} = \left( m + \frac{J}{\rho^2} \right) \ddot{L} + m\ddot{R} \sin \theta - mL\dot{\theta}^2 - mg \cos \theta \\ 0 &= 2m\dot{L}\dot{\theta} + mL\ddot{\theta} + m\ddot{R} \cos \theta + mg \sin \theta \end{aligned} \quad (36)$$

We considering the following operating point parameter values ([munkaponti paraméter értékek](#)):

$$R_0, \quad L_0, \quad \Theta_0 = 0, \quad F_0 = 0, \quad T_0 = mg\rho \quad (37)$$

Than we center the input and state variables:

$$R = R_0 + r, \quad L = L_0 + l, \quad F = F_0 + f, \quad T = T_0 + \tau \quad (38)$$

Substituting these expressions into (36), we get:

$$\underbrace{f - 2m\dot{l}\dot{\theta} \cos \theta + m\dot{\theta}^2(L_0 + l) \sin \theta}_{b_1} = \underbrace{\ddot{r}}_{a_{11}} \underbrace{m \sin \theta}_{a_{12}} + \underbrace{\ddot{\theta}}_{a_{12}} \underbrace{m(L_0 + l) \cos \theta}_{a_{13}} \quad (39)$$

$$\underbrace{M(L_0 + l)\dot{\theta}^2 - mg(1 - \cos \theta) - \frac{\tau}{\rho}}_{b_2} = \underbrace{\ddot{l}}_{a_{21}} \underbrace{\left( m + \frac{J}{\rho^2} \right)}_{a_{21}} + \underbrace{\ddot{r}}_{a_{22}} \underbrace{m \sin \theta}_{a_{22}} \quad (40)$$

$$\underbrace{-2m\dot{l}\dot{\theta} - mg \sin \theta}_{b_3} = \underbrace{\ddot{r}}_{a_{32}} \underbrace{m \cos \theta}_{a_{32}} + \underbrace{\ddot{\theta}}_{a_{33}} \underbrace{m(L_0 + l)}_{a_{33}} \quad (41)$$

Let us introduce the following state and input variables:

$$\begin{array}{ccc} x_1 = l & x_3 = r & x_5 = \theta \\ x_2 = \dot{l} & x_4 = \dot{r} & x_6 = \dot{\theta} \end{array} \rightarrow x = \begin{pmatrix} x_1 \\ \dots \\ x_6 \end{pmatrix} \quad \begin{array}{ccc} u_1 = f & & \\ u_2 = \tau & \rightarrow & u = \begin{pmatrix} u_1 \\ u_2 \end{pmatrix} \end{array} \quad (42)$$

Then, the resulting nonlinear state equation is the following

$$\begin{pmatrix} \dot{x}_1 \\ \dot{x}_3 \\ \dot{x}_5 \end{pmatrix} = \begin{pmatrix} x_2 \\ x_4 \\ x_6 \end{pmatrix} = \begin{pmatrix} f_1(x, u) \\ f_3(x, u) \\ f_5(x, u) \end{pmatrix}, \quad \begin{pmatrix} \dot{x}_2 \\ \dot{x}_4 \\ \dot{x}_6 \end{pmatrix} = \begin{pmatrix} \dot{l} \\ \ddot{r} \\ \ddot{\theta} \end{pmatrix} = \begin{pmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & 0 \\ 0 & a_{32} & a_{33} \end{pmatrix}^{-1} \begin{pmatrix} b_1 \\ b_2 \\ b_3 \end{pmatrix} = \begin{pmatrix} f_2(x, u) \\ f_4(x, u) \\ f_6(x, u) \end{pmatrix} \quad (43)$$

Computing the matrix inverse, we get

$$\begin{pmatrix} f_2(x, u) \\ f_4(x, u) \\ f_6(x, u) \end{pmatrix} = \frac{1}{a_{11}a_{22}a_{33} - a_{12}a_{21}a_{33} + a_{13}a_{21}a_{32}} \begin{pmatrix} a_{13}a_{32}b_2 - a_{12}a_{33}b_2 - a_{13}a_{22}b_3 + a_{22}a_{33}b_1 \\ a_{13}a_{21}b_3 + a_{11}a_{33}b_2 - a_{21}a_{33}b_1 \\ a_{11}a_{22}b_3 - a_{12}a_{21}b_3 - a_{11}a_{32}b_2 + a_{21}a_{32}b_1 \end{pmatrix} \quad (44)$$

Therefore, the nonlinear state-space model can be written as follows:

$$\dot{x} = F(x, u), \quad \text{where} \quad F(x, u) = \begin{pmatrix} f_1(x, u) \\ f_2(x, u) \\ f_3(x, u) \\ f_4(x, u) \\ f_5(x, u) \\ f_6(x, u) \end{pmatrix} \quad (45)$$

In order to get a linearized model (in the operating point  $(x_0 = 0, u_0 = 0)$ ), we considering the second order Taylor polynomial of  $F(x, u)$ :

$$F(x, u) \simeq F(x_0, u_0) + \left[ \frac{\partial F(x, u)}{\partial x} \right]_{\substack{x=0 \\ u=0}} (x - x_0) + \left[ \frac{\partial F(x, u)}{\partial u} \right]_{\substack{x=0 \\ u=0}} (u - u_0) \quad (46)$$

Since  $F(x_0, u_0) = 0$ , the linearized model is  $\dot{x} = Ax + Bu$ , where:

$$A = \left[ \frac{\partial F(x, u)}{\partial x} \right]_{\substack{x=0 \\ u=0}} = \begin{pmatrix} 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{gm}{M} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & -\frac{g(M+m)}{L_0M} & 0 \end{pmatrix}, \quad B = \left[ \frac{\partial F(x, u)}{\partial u} \right]_{\substack{x=0 \\ u=0}} = \begin{pmatrix} 0 & 0 \\ 0 & -\frac{\rho}{m\rho^2+J} \\ 0 & 0 \\ \frac{1}{M} & 0 \\ 0 & 0 \\ -\frac{1}{L_0M} & 0 \end{pmatrix} \quad (47)$$

As one can immediately observe, we obtained two decoupled subsystems in the linearized model:

$$\begin{aligned} \dot{\xi}_1 &= A_1 \xi_1 + B_1 \tau, \quad \text{where} \quad A_1 = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix}, \quad B_1 = \begin{pmatrix} 0 \\ -\frac{\rho}{m\rho^2+J} \end{pmatrix} \\ \dot{\xi}_2 &= A_2 \xi_2 + B_2 f, \quad \text{where} \quad A_2 = \begin{pmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & \frac{gm}{M} & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & -\frac{g(M+m)}{L_0M} & 0 \end{pmatrix}, \quad B_2 = \begin{pmatrix} 0 \\ \frac{1}{M} \\ 0 \\ -\frac{1}{L_0M} \end{pmatrix} \end{aligned} \quad (48)$$

The new state vectors  $\xi_1$  and  $\xi_2$  are the following:

$$\xi_1 = \begin{pmatrix} l \\ j \end{pmatrix} = \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}, \quad \xi_2 = \begin{pmatrix} r \\ \dot{\theta} \end{pmatrix} = \begin{pmatrix} x_3 \\ x_4 \\ x_5 \\ x_6 \end{pmatrix} \quad (49)$$

TODO

$$\begin{aligned}
 \theta(\varphi) - D &= \int \sqrt{\frac{C^2}{\sin^4 \varphi - C^2 \sin^2 \varphi}} d\varphi = \int \frac{\frac{1}{\sin^2 \varphi}}{\sqrt{\frac{1}{C^2} - \frac{1}{\sin^2 \varphi}}} d\varphi = \int \frac{\frac{1}{\sin^2 \varphi}}{\sqrt{\frac{1}{C^2} - \frac{\sin^2 \varphi + \cos^2 \varphi}{\sin^2 \varphi}}} d\varphi \\
 &= \int \frac{\frac{1}{\sin^2 \varphi} d\varphi}{\sqrt{\frac{1}{C^2} - 1 - \cot^2 \varphi}} = \int \frac{(\cot \varphi)' d\varphi}{\sqrt{B^2 - \cot^2 \varphi}} = \int \frac{dv}{\sqrt{B^2 - v^2}} = \arccos\left(\frac{v}{B}\right) \quad (50) \\
 &= \arccos\left(\frac{\cot \varphi}{\sqrt{\frac{1}{C^2} - 1}}\right) = \arccos\left(\frac{|C| \cot \varphi}{\sqrt{1 - C^2}}\right)
 \end{aligned}$$

$v := \cot \varphi$

tehát az általános megoldás:

$$\theta(\varphi) = D + \arccos\left(\frac{|C| \cot \varphi}{\sqrt{1 - C^2}}\right) \quad (51)$$